

**GUJARAT TECHNOLOGICAL UNIVERSITY****M. E. I<sup>ST</sup> Semester–Remedial Examination – July- 2011****Subject code: 711101****Subject Name: Advanced thermodynamics and Heat Transfer****Date: 07/07/2011****Time: 10:30 am – 01:00 pm****Total Marks: 60****Instructions:**

1. Attempt all questions.
2. Make suitable assumptions wherever necessary.
3. Figures to the right indicate full marks.
4. Students are allowed to use their own charts & tables.

- Q.1** (a) What is law of corresponding state? Explain Van Der Waal's equation as law of corresponding state. **06**
- (b) Derive the equation for heat transferred and maximum temperature available for a cylinder with uniform internal heat generation in one dimension. **06**

- Q.2** (a) Air expands through a turbine from 500 kPa, 520°C to 100 kPa, 300°C. During the expansion 10 kJ/kg of heat is lost to the surroundings which is at 98kPa, 20°C. Neglecting the K.E. and P.E. changes, determine per kg/s of air (1) the decrease in availability, (b) the maximum work and (3) the irreversibility. For air, take  $c_p = 1.005$  kJ/kg-K,  $h = c_p T$ . **06**
- (b) Define (i) compressibility factor (ii) reduced co-ordinates. Also With neat sketch explain general compressibility chart. **06**

**OR**

- (b) Explain in detail different types of Phase Equilibrium diagram on p-T surface, h-s surfaces, T-s surfaces. **06**
- Q.3** (a) Define Gibbs & Helm Holtz free energy and Derive Maxwell relations for internal energy (u), enthalpy (h), Gibbs energy (g) and Helm Holtz energy (f). **06**
- (b) Define unsteady flow process. Explain bottle filling and emptying processes. **06**

**OR**

- Q.3** (a) State Kelvin statement for second law and with neat sketch prove that for cyclic process  $\oint \delta Q/T \leq 0$  **06**
- (b) During summer to cool water for drinking purpose 1 kg of ice at -2°C is mixed with 5 kg of water at 27°C in an insulated container which is subjected to atmospheric pressure of 1 bar. Calculate the temperature of the mixture and the change in entropy for the spontaneous process. Assume specific heat of ice as 2.093 kJ/kg – K and the latent heat of ice as 335 kJ/kg. take  $C_p$  of water as 4.18 kJ/kg-K. **06**

- Q.4** (a) Explain the relaxation technique to solve the heat transfer problem on two-dimensional steady state conduction with neat sketch. **06**
- (b) A 6 mm thick stainless steel plate ( $\rho=7800$  kg/s/m<sup>3</sup>,  $c= 460$  J/kg-K,  $k = 55$ W/m°C) is used to form the nose section of missile. It is held initially at a uniform temperature of 30°C. When the missile enters the denser layers of the atmosphere at a very high velocity the **06**

effective temperature of air surrounding the nose region attains the value  $2150^{\circ}\text{C}$ ; the surface convective heat transfer co-efficient is estimated as  $3395 \text{ W/m}^2 \text{ }^{\circ}\text{C}$ . If the maximum metal temperature is not to exceed  $1100^{\circ}\text{C}$ . Determine: (i) maximum permissible time in these surrounding (ii) Inside surface temperature under these conditions.

**OR**

- Q.4** (a) Derive the Von Karman momentum equation for boundary layer flow. **06**  
 (b) Derive the equation for Reynold's analogy between fluid friction & Newton's Law of Viscosity. **06**

- Q.5** (a) State Lambert's cosine law. **06**  
 Calculate the following for an industrial furnace in the form of a black body and emitting radiation at  $2500^{\circ}\text{C}$  :  
 (i) Monochromatic emissive power at  $1.2 \mu\text{m}$  length,  
 (ii) Wavelength at which the emission is maximum, (iii) Maximum emissive power, (iv) total emissive power.  
 Take  $C_1 = 0.3742 \times 10^{-15} \text{ Wm}^4/\text{m}^2$  &  $C_2 = 1.4388 \times 10^{-2} \text{ mK}$   
 (b) Derive the equation for interchange factor  $f_{1-2}$  from surface 1 to 2 for a two infinite parallel plates. **06**

Where  $f_{1-2} = \frac{1}{1/\epsilon_1 + 1/\epsilon_2 - 1}$  with usual notations.

**OR**

- Q.5** (a) Define fin efficiency, fin effectiveness, overall effectiveness, Biot number, eddy diffusivities  $\epsilon_M$  and  $\epsilon_H$ . **05**  
 (b) A plane wall is provided with rectangular fin ( $k = 150 \text{ W/mK}$ ) having profile area  $0.9 \text{ cm}^2$ . The ambient condition is  $26^{\circ}\text{C}$  and unit surface conductance is  $50 \text{ W/m}^2\text{K}$ . If the fin base temperature is maintained at  $200^{\circ}\text{C}$ , calculate the maximum heat dissipated by fins per unit width of the fin. **07**

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