

**GUJARAT TECHNOLOGICAL UNIVERSITY****M. E. - SEMESTER – II • EXAMINATION – WINTER • 2013****Subject code: 1722101****Date: 24-12-2013****Subject Name: Design of Heat Exchange Equipments****Time: 10.30 am – 01.00 pm****Total Marks: 70****Instructions:**

1. Attempt all questions.
2. Make suitable assumptions wherever necessary.
3. Figures to the right indicate full marks.

**Q.1 (a)** Explain the factors to be considered while selecting heat exchangers? **07**

**(b)** For a balanced heat exchanger ( $\frac{C_{\min}}{C_{\max}} = 1$ ), show that **07**

1. For Parallel flow,  $\varepsilon = \frac{1 - e^{-2NTU}}{2}$

2. For counter flow,  $\varepsilon = \frac{NTU}{1 + NTU}$

**Q.2 (a)** Give detailed classification of heat exchanger based on construction and flow arrangement of fluid? **07**

**(b)** Draw the temperature profile along the length of heat exchanger for following cases (1)  $C_h \rightarrow \infty$  (2)  $C_h = C_c$  and (3)  $C_c \rightarrow \infty$  **07**

**OR**

**(b)** 1. When is a heat exchanger classified as being compact? Do you think a double pipe heat exchanger can be classified as a compact heat exchanger? **07**

2. In the heat transfer relation  $\dot{Q} = UAF\Delta T_{lm}$  for a heat exchanger, What is the quantity F called? What does it represent? Can F be greater than one?

**Q.3 (a)** Explain the operation of a plate heat exchanger? What are its applications? **07**

**(b)** Explain the approximate design of plate heat exchanger to suit preliminary sizing of the plate units for a given heat duty? **07**

**OR**

**Q.3 (a)** Explain passes and flow arrangement in gasket plate heat exchanger. State its applications. **07**

**(b)** Give classification of evaporator in detail? How they are different than other heat exchanger? Name the three recent correlations for intube flow boiling refrigerants? **07**

**Q.4 (a)** Write two important functions of baffle and explain different type and geometry of baffles used in a shell and tube heat exchanger? **07**

**(b)** A heat exchanger is to be designed to heat raw water by the use of condensed water at 67 °C and 0.2 bar which will flow in the shell side with a mass flow rate of 50,000 kg/h. The heat will be transferred to 30,000 kg/h of city water coming from a supply at 17 °C. Water outlet temperature should not be less than 40 °C. A fouling resistance of 0.000176 m<sup>2</sup>.K/W is suggested. Calculate the overall heat transfer coefficient for fouled as well as clean surfaces and length of tube for 2-P shell and tube heat exchanger using “Kern method”. Use following specifications and Correlations; **07**

|                                                                         |                    |                                                                                                                                        |                       |
|-------------------------------------------------------------------------|--------------------|----------------------------------------------------------------------------------------------------------------------------------------|-----------------------|
| Tube side specification                                                 |                    | Shell side specification                                                                                                               |                       |
| O.D. = 19 mm                                                            |                    | Shell diameter = 0.39 m                                                                                                                |                       |
| I.D. = 16 mm                                                            |                    | Pitch size = 0.024                                                                                                                     |                       |
| No. of tubes = 124                                                      |                    | Baffle spacing = 0.25 m                                                                                                                |                       |
| Tube layout- Square Pitch                                               |                    | $Nu = 0.36 \left( \frac{D_e G_s}{\mu} \right)^{0.55} \left( \frac{\mu C_p}{k} \right)^{1/3} \left( \frac{\mu_b}{\mu_w} \right)^{0.14}$ |                       |
| K <sub>tube</sub> = 60 W/m <sup>2</sup> K                               |                    |                                                                                                                                        |                       |
| $Nu_b = \frac{(f/2)(Re_b - 1000)Pr}{1 + 12.7(f/2)^{1/2}(Pr^{2/3} - 1)}$ |                    |                                                                                                                                        |                       |
| Where;<br>$f = (1.58 \ln Re - 3.28)^{-2}$                               |                    |                                                                                                                                        |                       |
| Properties                                                              | Units              | Tube side                                                                                                                              | Shell side            |
| ρ                                                                       | Kg/m <sup>3</sup>  | 996.8                                                                                                                                  | 983.2                 |
| C <sub>p</sub>                                                          | J/kg K             | 4179                                                                                                                                   | 4184                  |
| μ                                                                       | N.s/m <sup>2</sup> | 8.2×10 <sup>-4</sup>                                                                                                                   | 4.67×10 <sup>-4</sup> |
| k                                                                       | W/m K              | 0.610                                                                                                                                  | 0.652                 |
| Pr                                                                      | -                  | 5.65                                                                                                                                   | 3.00                  |
|                                                                         |                    | $\mu_w = 6.04 \times 10^{-4} \text{ N.s/m}^2$                                                                                          |                       |

**OR**

- Q.4 (a)** Draw and explain various shell designs suggested by the “TEMA standard” for the shell and tube heat exchanger? **07**
- (b)** The condenser of a large steam power plant is a shell and tube heat exchanger having a single shell and 30,000 tubes, with each tube making two passes. The tubes are thin walled with 25 mm diameter and steam condenses on the outside of the tubes with  $h_o = 11 \text{ kW/m}^2 \text{ K}$ . The cooling water flowing through the tubes is 30,000 kg/s and the heat transfer rate is 2 GW. Water enters at 20 °C while steam condenses at 50 °C. Find the length of the tubes in one pass. Properties of water at 27 °C are  $C_p = 4.18 \text{ kJ/kg K}$ ,  $\mu = 855 \times 10^{-6} \text{ N.s/m}^2$ ,  $k = 0.613 \text{ W/m K}$  and  $Pr = 5.83$ . The flow of water is turbulent, use Dittus-Boelter equation towards water side:  $Nu = 0.023 Re^{0.8} Pr^{0.4}$  **07**

- Q.5 (a)** Explain the strategies to control fouling? **07**
- (b)** Explain the designed of coal fired furnace? **07**

**OR**

- Q.5 (a)** What is hairpin heat exchanger? Define hydraulic ( $D_h$ ) and Equivalent ( $D_e$ ) diameters for bare inner tube and annulus of a hairpin heat exchanger. **07**
- (b)** Explain rotary and fixed matrix regenerators with their applications. **07**

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